DEFINING THE STABILITY LIMITS OF A MANUAL WHEELCHAIR WITH ADJUSTABLE SEAT AND BACKREST

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BACKGROUND

Over 60% of active manual wheelchair users experienced falls due to instability over a three year period (Chen et al., 2011). Such incidents occasionally resulted in traumatic brain injury or bone fractures (Opalek et al., 2009). Clearly wheelchair stability is important, but it is not the only design consideration.

Manual wheelchairs must also be easy to push, i.e. maneuverable. This comes at the cost of reduced stability (Brubaker, 1986; Tomlinson, 2000). Maneuverability is improved by increasing the load on the rear wheels (normally done by moving the rear axle position forward). In contrast, the wheelchair is more stable when the load is distributed between the front and rear wheels. A compromise is found between the two objectives, with the optimal configuration dependant on the specific use case.

Adjustable "on the fly" or dynamic seating allows users to change their wheelchair seat configuration throughout the day (Borisoff and McPhail, 2011). Such changes have been identified by RESNA as important for health reasons, easing transfers, improving reach, and enhancing independence (Arva et al., 2009). Dynamic seating changes may move the centre of gravity of the system, affecting the maneuverability and stability of the wheelchair. However, to date little quantitative research has been conducted on the extent of these effects.

PURPOSE

This study aimed to determine the effects of seat dump, backrest angle, rear axle position, and user position (i.e. offset between a user's hips and the backrest) on wheelchair stability and maneuverability, and to identify optimal seat configurations for sloped environments.

METHODS

Wheelchair simulation

To evaluate the stability of a wheelchair with a range of seat and backrest configurations, a rigid body dynamic model of a manual wheelchair was developed using MADYMO software (TASS International, Netherlands). A 250lb ISO test dummy was modelled, which is the design limit of many manual wheelchairs.

The geometry of the simulation was created using a CAD model of an ultralight manual wheelchair with dynamic seating (Elevation[™], PDG Mobility, Canada) and validated using physical measurements of the same wheelchair. The wheelchair had 25" diameter wheels, 5" casters, a seat depth of 16", and a seat width of 16". The 250-lb test dummy was modeled to meet ISO 7176-11 standards.

Experimental testing

The wheelchair model was validated by comparing the stability of the simulation to that of a physical wheelchair. Static stability was tested in accordance with ISO 7176.1. An engine hoist was used to lift a platform with a block fixed at the bottom to prevent the wheelchair from rolling down (Figure 1). A 3D motion capture system (Qualisys, Sweden) determined both the time at which the wheelchair started tipping, and the corresponding angle of the ramp.

A full-factorial array of three seat and three backrest positions was tested.



Figure 1: Static stability test setup, showing ramp lifted into a slope with engine hoist and wheelchair stopped from rolling with a block.

Configurations ranged from a seat angle of -13° and vertical backrest (Figure 2b) to a seat angle of 16° and backrest reclined 35° (Figure 2c). Each configuration was tested three times for both forwards and backwards stability. In all cases, there was less than 2° difference between the simulated and experimental tip results.

Analysis

The simulations were run for full-factorial combinations of five backrest angles (-5° to 35°), seat dumps (-10° to 20°), rear axle positions (0cm to 20cm forward from the backrest), and offset distances between the user and the backrest (0cm to 8cm). The

reaction forces on each wheel and the tip angles were recorded. The ratio of load on the rear wheels were calculated using MATLAB (Mathworks Inc., USA). This metric related the centre of gravity (CoG) of the wheelchairuser system to its performance, with a higher ratio indicating a more maneuverable but less stable configuration.

RESULTS

The backward stability of the wheelchair increased when the rear axles were positioned further back, when the backrest was more upright, and for greater user offsets (Figure 3). These changes moved the CoG of the system forward relative to the rear wheels. The magnitude of stability changes due to the backrest angle was also dependent on all other configuration variables. No other parameter had significant dependencies.

The angle of the backrest had the greatest effect on stability when the rear axle was moved forward, there was no user offset, and the seat was fully lowered. For a rear axle position of 10cm and no offset, a stability change of over 20° could be achieved just by changing the backrest (left middle panel, Figure 3).



Figure 2: Wheelchair configurations showing (a) the definitions of all variables being tested,(b) the wheelchair, including dummy, in a fully elevated position with the backrest upright,(c) the wheelchair in a fully lowered position with the backrest fully back.



Figure 3: Rear stability changes for different seat and backrest angles. Panels are grouped horizontally by rear axle position (5cm, 10cm, 15cm), and vertically by the user offset (0cm, 5cm).

In general, a lower rear seat corresponded to a small increase in stability, with greater effects for more extreme seat angles. For certain more stable configurations, the effect of seat dump on stability was reversed.

On smaller ramp slopes the seat angle had little effect on wheelchair performance, while back angle had a greater effect (Figure 4). However, on steeper slopes and at more extreme settings, seat dump became a greater factor. While facing downhill the chair was most stable (less maneuverable) with the seat fully lowered and backrest reclined. facing When uphill, the stability was predominately affected by backrest angle. On steep uphill slopes (1:6), the wheelchair became unstable for any backrest angle greater than 20°.

DISCUSSION

A wheelchair should be stable enough so that it does not tip over in the user's environment, but any more stability than necessary may impact performance and maneuverability (Tomlinson, 2000). Therefore, the load ratio on the rear wheels should be as high as possible without causing instability, (which always occurs at a ratio of 1).

For a fixed frame seating configuration, the stability and maneuverability are changed by the positioning of the rear axles and the posture of the user. Though the rear axle positioning has the greater effect of these two variables. our results show that user positioning is not inconsequential. Therefore,



igure 4. Load ratios on the rear wheels for different seat and backlest configurations when the rear axle position is held constant at 10cm and there is no user offset. Each panel shows the wheelchair on a different slope. A higher ratio represents a greater percentage of the load on the drive wheels, indicating better maneuverability. However, a load ratio approaching 1 indicates instability.

user posture should be considered when initially configuring a wheelchair and when instructing a new user. For example, if a user naturally slouches in a way that their lower back is 1cm offset from the base of the backrest, the rear axles could be moved forward about 0.4cm to maintain the stability and maneuverability of the wheelchair system (keeping all other variables constant). In addition, if a user moves forward in their seat the wheelchair will become more stable but less efficient for wheeling performance.

For wheelchairs with the capability, onthe-fly changes to seat and backrest configurations allow the wheelchair to be therefore more tippy, and more maneuverable, for a set initial configuration. For a fixed rear axle position and user offset, our simulations showed changes to the seat and backrest positions enable the wheelchair stability to vary by up to 22°. Backrest position has the greatest effect on stability and maneuverability; however, seat dump angle also affects the performance on steeper slopes.

Using dynamic changes to the wheelchair configuration, it is also possible to maintain the same front/rear wheel load distribution, and therefore maneuverability, of the wheelchair when it is on a slope. Figure 4 shows that when a wheelchair is set up to be stable on level ground, a rear load ratio of 0.75 (as used by Tomlinson, 2000) can be maintained for any slope between +9.5° and -9.5° by adjusting the seat and backrest angles. This is well within the wheelchair ramp standard of 1:12, or 4.8°. The backrest angle, rather than the seat dump, is the main enabler for this range.

Maintaining an optimal rear wheel load ratio could improve wheeling capabilities and safety in the community. For instance, when wheeling uphill, a backrest adjusted forward would provide support to users leaning into the slope; similarly, when traveling downhill, a backrest adjusted to more recline would provide the user with balanced trunk support and wheeling stability (Borisoff and McPhail, 2011; Hong et al., 2011), and obviate the need to be in a "wheelie".

Limitations

A major limitation of the model is that it only looked at static stability, which does not completely reflect real world wheelchair use. As the user was modelled on an ISO dummy, it will also only represent a certain percentage of the wheelchair user population.

CONCLUSION

Rear axle position has the greatest effect on wheelchair stability and maneuverability. The backrest angle was the next most influential factor, and had significant dependencies on each of the other variables. By adjusting back and seat angles, stability changes of over 20° can be achieved.

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